

**UNIVERSITY OF BOLTON**

**OFF CAMPUS DIVISION**

**WESTERN INTERNATIONAL COLLEGE**

**BENG (HONS) MECHANICAL ENGINEERING**

**SEMESTER ONE EXAMINATION 2023/24**

**MECHANICS OF MATERIALS AND MACHINES**

**MODULE NO: AME5012**

Date: Tuesday 09 January 2024

Time: 10:00 AM – 12:00 PM

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**INSTRUCTIONS TO CANDIDATES:**

There are FIVE questions on this paper.

Answer any FOUR questions

All questions carry equal marks.

Marks for parts of questions are shown in brackets.

Electronic calculators may be used provided that data and program storage memory is cleaned prior to the examination.

**CANDIDATES REQUIRE:**

Formula Sheet (attached)  
Graph Paper

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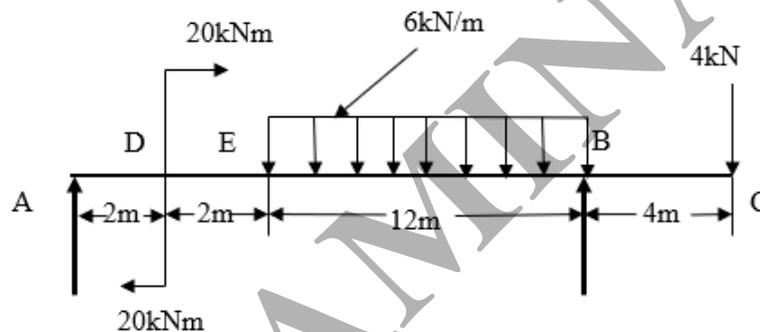
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**Q1.** For the simply supported overhanging beam AC of length 10m which is supported at A and B, shown in **Figure Q1**, use Macaulay's method to determine:

- the slope and deflection equations for the beam (16 marks)
- the slope at A and B (6 marks)
- the deflection at D (3 marks)

Take Flexural rigidity,  $E = 2 \times 10^5 \text{ N/mm}^2$  ;  $I = 10^8 \text{ mm}^4$

**Total 25 marks**



**Figure Q1** simply supported overhanging beam

**Q2.** A damped spring-mass system with mass  $m = 15\text{kg}$ , spring stiffness  $k = 40 \text{ kN/m}$  and damping ratio  $\xi = 0.25$  is subjected to a simple harmonic disturbing force of  $70 \cos 25t$  newtons. Determine :

- the amplitude and phase lag of the steady state vibrations (12 marks)
- the amplitude of the steady state vibration when  $\omega = \omega_n$  (4 marks)
- the frequency of the varying force, which will give maximum amplitude and the value of this maximum amplitude. (9 marks)

**Total 25 marks**

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**Q3.** An element of material is subjected to a two dimensional stress system as shown in **Figure Q3**.

a) Using a scale of 1 cm = 10 MPa, construct Mohr's stress circle (8 marks)

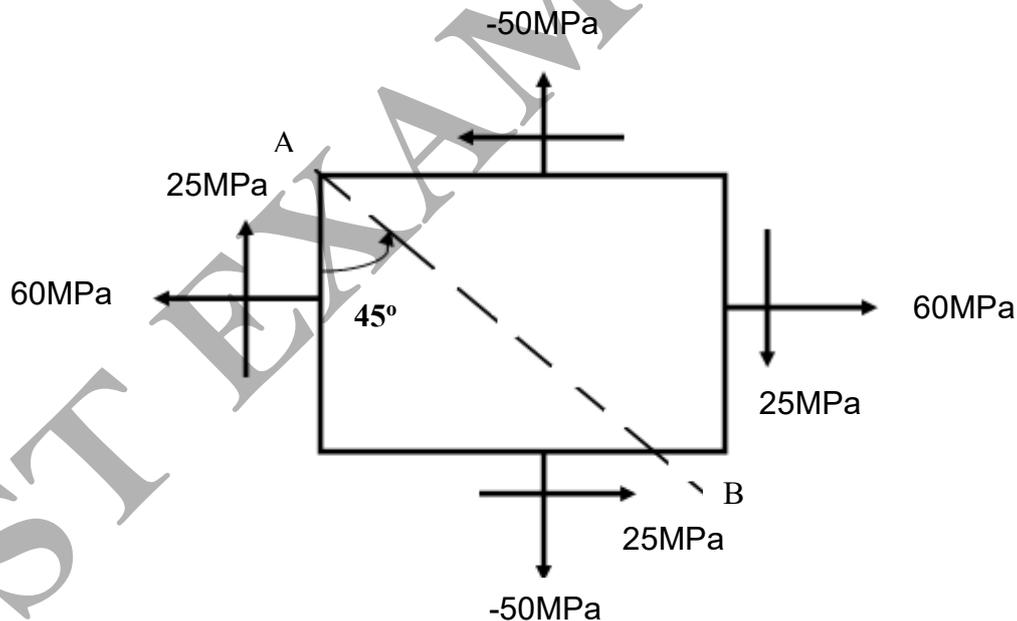
and hence determine :

(i) The magnitude of the principal stresses (3 marks)

(ii) The magnitude of the maximum shear stress (3 marks)

(iii) The normal and shear stresses on the plane AB (3 marks)

b) Confirm the magnitude of the principal stresses by calculation. (8 marks)



**Figure Q3.** Two-dimensional stress system

**Total 25 marks**

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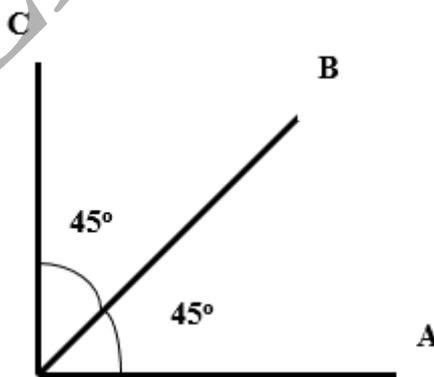
**Q4.** A thick-walled cylinder is subjected to an internal pressure of 60 MPa. If the cylinder internal diameter is 11 cm and external diameter is 19 cm, determine the following:

- a) the circumferential (hoop) stress at the inside and outside radii (8 marks)
- b) the longitudinal stress across the wall section (3 marks)
- c) the change in the internal diameter and the change in length due to the internal pressure if the original length is 4m. (6 marks)
- d) Sketch the distribution of the circumferential and radial stresses across the wall section also indicating the longitudinal stress. (8 marks)

Take Modulus of elasticity,  $E = 200\text{GPa}$  & Poisson's ratio,  $\nu = 0.25$

**Total 25 marks**

**Q5.** Figure Q5 shows a  $45^\circ$  rectangular strain gauge rosette which is bonded to the surface of a steel structural member.



**Figure Q5**  $45^\circ$  rectangular strain gauge rosette

**Question 5 continued over...**  
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**Question 5 contd.**

When the structure is loaded the strain readings are :

Gauge A :  $-434 \times 10^{-6}\text{m}$

Gauge B :  $-144 \times 10^{-6}\text{m}$

Gauge C :  $538 \times 10^{-6}\text{m}$

- a) Construct and label Mohr's strain circle to a scale of  $1\text{cm} = 100 \times 10^{-6}$   
(10 marks)
- b) Superimpose Mohr's stress circle onto the strain circle. (5 marks)
- c) From the two circles, determine :
- (i) The principal strains (2 marks)
  - (ii) The principal stresses (4 marks)
- d) Verify the magnitudes of the principal stresses using the appropriate formula. (4 marks)

Take Modulus of elasticity,  $E = 200 \text{ GPa}$ , Poisson's ratio  $\nu = 0.3$

**Total 25 marks**

**END OF QUESTIONS**

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### FORMULA SHEET

#### Deflection

$$EI \frac{d^2y}{dx^2} = M$$

#### Complex Stress

$$\sigma_{\theta} = \frac{\sigma_x + \sigma_y}{2} + \left( \frac{\sigma_x - \sigma_y}{2} \right) \cos 2\theta + \tau \sin 2\theta$$

$$\tau_{\theta} = \left( \frac{\sigma_x - \sigma_y}{2} \right) \sin 2\theta - \tau \cos 2\theta$$

$$\tan 2\theta_p = \frac{2\tau}{\sigma_x - \sigma_y}$$

#### Complex Strain

$$\text{Radius of stress circle} = \frac{(1-\nu)}{(1+\nu)} \times \text{Radius of strain circle}$$

$$\text{Stress circle} = \frac{E}{(1-\nu)} \times \text{strain scale}$$

$$\sigma_1 = \frac{E(\epsilon_1 + \nu\epsilon_2)}{1-\nu^2} \quad \sigma_2 = \frac{E(\epsilon_2 + \nu\epsilon_1)}{1-\nu^2}$$

#### Thick Cylinder

Lame' Equations

$$\sigma_c = A + \frac{B}{r^2}, \quad \sigma_R = A - \frac{B}{r^2}$$

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Strain Format

$$\varepsilon_x = +\frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

Strain along any angle

$$\varepsilon_\theta = \varepsilon_x \sin 2\theta + \varepsilon_y \cos 2\theta + \gamma_{xy} \sin \theta \cos \theta$$

Vibrations

$$f_n = \frac{\omega_n}{2\pi} \quad \omega_n = \sqrt{\frac{k}{m}}$$

Damped  $f_d = \frac{\omega_d}{2\pi} \quad \omega_d = \omega_n \sqrt{1 - \xi^2}$

Log Decrement  $\ell_n \frac{x_1}{x_r} = \frac{2\pi(r-1)\xi}{\sqrt{1-\xi^2}}$

Critical Damping  $C_c = 2m\omega_n \quad \xi = \frac{C}{C_c}$

Forced  $X_0 = \frac{F/K}{\sqrt{(2\xi r)^2 + (1-r^2)^2}} \quad \phi = \tan^{-1} \frac{2\xi r}{1-r^2}, \quad r = \frac{\omega}{\omega_n}$

$$r_{\text{res}} = \sqrt{1 - 2\xi^2}, \quad r_{\text{res}} = \frac{\omega_{\text{res}}}{\omega_n}$$

Transmissibility  $F_T = \sqrt{(kX_0)^2 + (c\omega X_0)^2}$